

A Zero-Emission Sustainable Landfill-Gas-To-Wire Oxyfuel Process: Bioenergy with Carbon Capture and Sequestration

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Abstract

Landfill-gas basically consists of methane and carbon dioxide and its reclamation is mandatory for better waste utilization and low greenhouse-gas emissions to transition towards renewable-energy matrices. Power generation or landfill-gas-to-wire is a solution for landfill-gas utilization with electricity-supply benefits skipping complex purification/transportation steps. As municipal solid-waste is rich in biomass, landfill-gas-fired power generation, with carbon capture and sequestration, leads to negative emissions and, consequently, climate-change mitigation. This work investigates the feasibility of zero-emission landfill-gas-to-wire concepts with oxyfuel carbon sequestration against conventional landfill-gas-fired plant facing carbon charges. Economic analysis is supported by Aspen-HYSYS simulation assuming large-scale landfill-gas supply. Different gas-turbine pressure-ratios are economically sought for both conventional and oxyfuel power plants to establish most profitable configurations. The greatest net values of conventional and oxyfuel plants are respectively attained for combustion pressures around 8 and 20 bar. This indicates investment and compression costs pulling down gas-turbine pressure-ratio in the landfill-gas combined-cycle plant. At such conditions, carbon dioxide is captured at 0.875 kg/kWh entailing oxyfuel efficiency penalty of 9.2% based on landfill-gas lower heating value, and increasing the long-term break-even electricity price from US\$36/MWh to US\$104/MWh. Economic superiority of zero-emission oxyfuel-combined-cycle over conventional plant occurs for carbon taxes above US\$95/t.

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Keywords: Landfill-Gas; Power Generation; Oxy-Combustion; CO₂; Bioenergy; CCS.

Highlights

- Oxyfuel landfill-gas-to-wire is evinced as a viable and sustainable solution.
- Two different oxyfuel landfill-gas-to-wire are evaluated and economically compared.
- Optimization of gas-turbine combustor pressure executed from economic perspective.
- Proposed oxyfuel landfill-gas-to-wire has break-even power price of US\$0.104/kWh.
- Oxyfuel landfill-gas-to-wire overcomes ordinary landfill-gas-to-wire at US\$95/tCO₂.

1 **Abbreviations**

2 ASU Air Separation Unit; BECCS Bioenergy with Carbon Capture and Sequestration; CCS
3 Carbon Capture and Sequestration; CCU Carbon Capture and Utilization; CW Cooling-
4 Water; DCC Direct-Contact-Column; EOR Enhanced Oil Recovery; GT Gas-Turbine; HRSG
5 Heat-Recovery Steam-Generator; LCOE Levelized Cost of Electricity; LGTW Landfill-Gas-
6 to-Wire; LHV Lower Heating Value; PCC-MEA Aqueous-Monoethanolamine Post-
7 Combustion CO₂ Capture; RIOCC Regenerative Intercooled Oxy-Combustion Combined-
8 Cycle; SCOC-CC Semi-Closed Oxy-Combustion Combined-Cycle; ST Steam-Turbine;
9 TVR-2REB Top-Vapor Recompression Two-Reboiler Distillation; USD US-Dollar.

10

11 **Nomenclature**

12 *AP, CEPCI* : Annual profit (USD/y), chemical engineering plant cost index
13 *c_p, c_v* : Specific heat capacities at constant pressure and constant volume (kJ/kgK)
14 *COM, CUT* : Costs of manufacturing, cost of utilities (USD/y)
15 *E_{sep}* : ASU specific power consumption (kWh/tO₂)
16 *F* : Molar flow rate (kmol/h)
17 *FCI, GAP* : Fixed capital investment (USD), Gross annual profit (USD/y)
18 *GWP_n* : Global warming potential for n years (kgCO₂eq/kg)
19 *i, ITR* : Annual interest rate, income tax rate (%)
20 *NPV, P, REV* : Net present value (USD), pressure (bar), revenues (USD/y)
21 *T, W* : Temperature (°C), mechanical power (kW)
22 *Y_k* : Molar fraction of component k

23 **Greek Symbols**

24 *γ* : Isentropic exponent of gas expansion/compression
25 *η* : Adiabatic efficiency of gas expansion/compression (%)

26 **Subscripts**

27 *Comb, CONV* : Combustor outlet, Conventional combined-cycle
28 *OXY-BECCS* : Oxyfuel-BECCS

29

30

1 **1. Introduction**

2 The Paris Agreement targets require strong mitigation efforts to all sectors, demanding
3 widespread adoption of renewable resources whenever possible and phasing out unmitigated
4 fossil-based power generation, in order to cut CO₂ emissions by ≈50% [1]. Consequently,
5 negative-emission initiatives play important role in climate-change stabilization [2] and
6 depend on widespread deployment of bioenergy with carbon capture and sequestration
7 (BECCS) enterprises [3], allowing to offset impacts from sectors where mitigation is more
8 expensive [4]. In this context, waste utilization avoids ecological and social issues associated
9 to biomass production for energy purposes [5]. In other words, waste-to-power BECCS
10 initiatives lead to socio-economic-environmental benefits [6] and provide fossil-fuel
11 displacement alternatives [7].

12 **1.1. Landfill-Gas**

13 Efficient municipal solid-waste management must explore waste valorization opportunities
14 and should promote circular economy principles, sending excess to landfills, and applying
15 thermochemical or biochemical conversion in a cost-effective sustainable manner [8].
16 Landfilling is the cheapest solid-waste management option [9], but without the recovery of
17 landfill-gas, considerable environmental/social impacts follow, besides loss of valorization
18 opportunities. Open dumps are the extreme example, being characterized by solid-waste
19 deposition on sites without due environmental care, severely polluting air, soil, and water,
20 besides offering explosion risks [10] and imposing threats to population health [9]. Both
21 leachate contamination and toxic compounds in landfill-gas are identified as health disorder
22 causes [11]. Thus, planned construction of sanitary landfills is conceived for mitigation of
23 such impacts and appropriate management of leachate and landfill-gas.

1 Landfill-gas production varies accordingly to landfill characteristics (e.g., size, age,
2 collection system) and solid-waste profile [12]. In volume dry-basis, a typical US landfill-gas
3 generated at steady conditions (i.e., after some operation years) has 50-55%CH₄, 45-50%CO₂
4 and 2-5% of other gases – e.g, N₂, H₂S, volatile organic compounds and siloxanes. At such
5 conditions, landfill-gas production is approximately stable for about 20 years [13]. Some
6 possible landfill-gas uses comprise electricity generation, biomethane commercialization,
7 heating, steam generation, leachate evaporation, and production of chemical/biochemical
8 derivatives [14]. The reduction of landfill emissions greatly results in climate-change
9 mitigation [15], as CH₄ has 28 times more global warming potential than CO₂ on weight basis
10 for 100 years (*GWP₁₀₀*), and 84 times for 20 years (*GWP₂₀*) [3]. In addition, further
11 greenhouse-gas emissions are abated if landfill-gas displaces fossil-fuel uses [15]. Thus, good
12 overall environmental performances depend on efficient landfill-gas collection systems [16].
13 If landfill-gas is not sent to useful purposes, it must be flared to eliminate CH₄, volatile
14 organics, and toxic/odorant compounds [17].

15 Landfill-gas utilization entails governance benefits due to improved urban energy security
16 and less dependence on distant electricity or natural gas suppliers [18]. Thus landfill-gas
17 power plants provide economic growth, creating revenues and jobs, while benefitting the
18 environment by reducing greenhouse-gas emissions and fossil-fuel utilization [19]. Electricity
19 tariffs are not supposed to increase, as the 2005 break-even price was showed to be lower
20 than 0.04 USD/kWh without government subsidies [20].

21 Landfill-gas power generation usually resorts to internal combustion engines of 0.1–3.0 MW
22 for small landfill-gas capacities of 50-960 scfm. Gas-turbines (GT) over 3 MW of power are
23 suitable when landfill-gas supply is sufficiently high and stable over 1050 scfm at ≈50%v/v
24 CH₄ [21]. Steam-turbines (ST) are also valid in large projects, with advantages of greater

1 flexibility and resiliency to landfill-gas contaminants, dismissing compression [22]. Some GT
2 advantages over internal combustion engines comprise greater corrosion resistance, compact
3 size, complete and cleaner combustion, and lower operation/maintenance costs [21].

4 **1.2. Bioenergy with Carbon Capture and Sequestration**

5 Beyond fossil-carbon capture and sequestration (CCS) and fossil-carbon capture and
6 utilization (CCU), geological sequestration of biogenic CO₂ has been suggested for effective
7 global warming mitigation [3]. BECCS allows negative life-cycle emissions due to CO₂
8 biofixation by photosynthesis and differently from afforestation and soil carbon sequestration
9 – without disturbances over terrestrial carbon stocks – BECCS contributes to the climate with
10 permanent CO₂ storage [5], promoting continuous drainage of atmospheric CO₂ [23]. BECCS
11 electricity generation also enhances flexibility and diversity of the energy portfolio [5],
12 besides improving regional energy independence [18]. Some countries, like US and Australia,
13 have been considered ready for BECCS deployment, considering their CCS and bioenergy
14 experiences. However, effective implementation will demand policy guidance [5], wherein
15 carbon taxation is a conceivable instrument [24].

16 On the other hand, widespread deployment of BECCS has techno-economic and social
17 challenges to overcome. Bioenergy often generates environmental impacts [25] related to
18 biomass cultivation and harvesting [26]. Other sustainability issues are storage availability
19 and possible competition for land and resources [2], potentially creating drivers for
20 deforestation, community displacements and biodiversity threats [6]. In some cases, positive
21 net CO₂ balance [27] and project feasibility are threatened by biomass production and
22 transportation costs and associated emissions [28]. However, these difficulties are attenuated
23 if BECCS is embedded in waste management and CCU is employed. A problem with the
24 latter is a lack of scale compatibility between CCU market and climate stabilization

1 requirements [29], so that a mix of CCS and many CCU routes should match the purpose of
2 CO₂ abatement. In this regard, the limited solution of enhanced oil recovery (EOR) is
3 currently the only way to add value to CO₂ at large-scales ($\approx 10^6$ t/y) [30]. Due to this
4 handicap, CO₂ geological sequestration is considered in this work.

5 **1.3. Oxy-Combustion Carbon Capture**

6 Among the power production routes with carbon capture, oxy-combustion allows better
7 environmental performance with profitability potential and greater net efficiencies, being the
8 only option capable of zero-emission power generation [31]. Oxy-combustion economic
9 competitiveness is heavily dependent on cost-effective air fractionation [32]. Currently,
10 cryogenic air separation is the most practical route for oxygen supply to large-scale oxy-
11 combustion systems [33]. The most efficient cryogenic air separation unit (ASU) for 95% mol
12 atmospheric gas oxygen production [32] is based on top-vapor recompression two-reboiler
13 cryogenic distillation (TVR-2REB) which requires 139 kWh/tO₂.

14 Another challenge for large-scale oxy-combustion implementation is the need of special
15 equipment operating in oxy-firing mode; i.e., oxyfuel GTs [34]. Nevertheless, oxyfuel power-
16 cycles have been proposed requiring reduced degree of modification in GT machinery. The
17 most intuitive is the so-called semi-closed oxy-combustion combined-cycle (SCOC-CC) [35].
18 Other CO₂-based cycles were developed without a bottoming steam-cycle, frequently
19 comprising intercooled compression to supercritical conditions (e.g., Allam-cycle) [34].
20 Another approach is water-based cycle, where condensate abates the combustion temperature
21 dismissing a gas-recycle; i.e., large CO₂-recycle compressors are substituted by water pumps.
22 Water-based cycles currently present the highest technology readiness level [34] accounting
23 for a successful demonstration project [36], but significantly lower efficiencies,
24 comparatively to gas-recycle configurations, have been reported [37]. Combining CO₂-

1 recycle and water-recycle is also possible (e.g., S-Graz cycle) and was reported to yield the
2 best results, though at the expenses of high complexity [38].

3 For a given combustor temperature (T_{Comb}) and steam-cycle conditions, there is an optimum
4 operational GT pressure-ratio [34]. According to Dahlquist et al. [39], this design parameter
5 is the linchpin of oxyfuel combined-cycles. For same T_{Comb} and exhaust temperature, oxyfuel
6 GTs uses higher pressure-ratios compared to conventional air-blown GTs [40], which
7 typically run at ≈ 20 bar combustion pressure (P_{Comb}) [41], while SCOC-CC GTs often run
8 with P_{Comb} ranges of 40-60 bar [40]. The underlying reason is the relatively low isentropic
9 exponent ($\gamma=c_p/c_v$) of CO₂-rich fluid in SCOC-CC [39], entailing lower temperature changes
10 through adiabatic compression/expansion [42]. The milder final temperatures (comparatively
11 to air) of CO₂-rich recycle adiabatic compression, favors P_{Comb} increases for higher power
12 outputs. However, high pressure-ratios increase turbo-machinery size and complexity, which
13 must be pondered over net efficiency gains [39]. The fluid behavior is also affected by the
14 higher density/compressibility and lower sound speed of CO₂-rich fluids comparatively to
15 N₂-rich fluids of usual GTs, changing GT annulus flow section [34]. The majority of works
16 addressing pressure-ratio of oxyfuel GTs only consider net power output perspective, so the
17 compression temperature raise is what generally limits P_{Comb} increase. The selection of the
18 most suitable pressure-ratio for equipment construction is then usually left to a subjective
19 decision accounting for turbo-machinery and heat-recovery steam-generator (HRSG) design
20 concerns. In SCOC-CC, the net efficiency curve is typically flat at optimum net efficiency
21 [43], so that the realistic best condition is slightly below the theoretical maximum [39]. In
22 this regard, Dahlquist et al. [39] obtained an optimum pressure-ratio of 45 for $T_{Comb}\approx 1340^\circ\text{C}$,
23 but considered that ≈ 34 should be the most advantageous solution. A further issue [41] is that
24 pressure-ratios above 30 are only common in aero-derivative GTs – where compressor design
25 favors aerodynamics and power efficiency – while industrial GTs adopt simpler designs

1 towards minimal fixed capital investment (*FCI*). In this work the optimal GT pressure-ratio
2 determination follows an economic formulation, taking into account equipment sizes and
3 their influence on profitability. Additionally, there is the fact that landfill-gas – contrarily to
4 natural gas – needs compression from atmospheric pressure, which also affects optimal
5 operating conditions.

6 **1.4. The Present Work**

7 An oxyfuel BECCS landfill-gas utilization is analyzed for zero-emission power generation,
8 entailing net removal of atmospheric CO₂ and avoidance of CH₄ emissions. The concept is
9 hereinafter named as landfill-gas-to-wire, analogously to natural gas gas-to-wire [44].

10 Currently, there are few literature works dealing with rigorous simulation of oxyfuel power
11 generation burning CO₂-rich fuel-gas. In addition, CO₂ emission taxation has not been
12 considered in landfill-gas-to-wire studies, though its recognized role for climate-change
13 mitigation [45]. In Chakroun and Ghoniem [37], the analyzed processes were not compared
14 to conventional natural gas combined-cycle and their fuel-gas composition differs from
15 landfill-gas CO₂ and H₂S ranges [46], significantly impacting power-cycle performance.

16 The present work demonstrates economic feasibility of oxyfuel BECCS-landfill-gas-to-wire
17 without overpriced electricity. Moreover, economically optimal oxyfuel GT P_{Comb} for a low-
18 pressure CO₂-rich fuel-gas is first-time used. These aspects are literature gaps that the present
19 work intends to fill. The proposed oxyfuel BECCS-landfill-gas-to-wire (LGTW-BECCS) is
20 proved not only environmentally superior, but also more profitable than landfill-gas air-fired
21 combined-cycle (LGTW-CONV) under CO₂ taxation. To the authors knowledge, energy and
22 economic assessments of oxyfuel landfill-gas combined-cycle LGTW-BECCS against
23 conventional air-blown LGTW-CONV never appeared before in the literature.

1 **2. Methods**

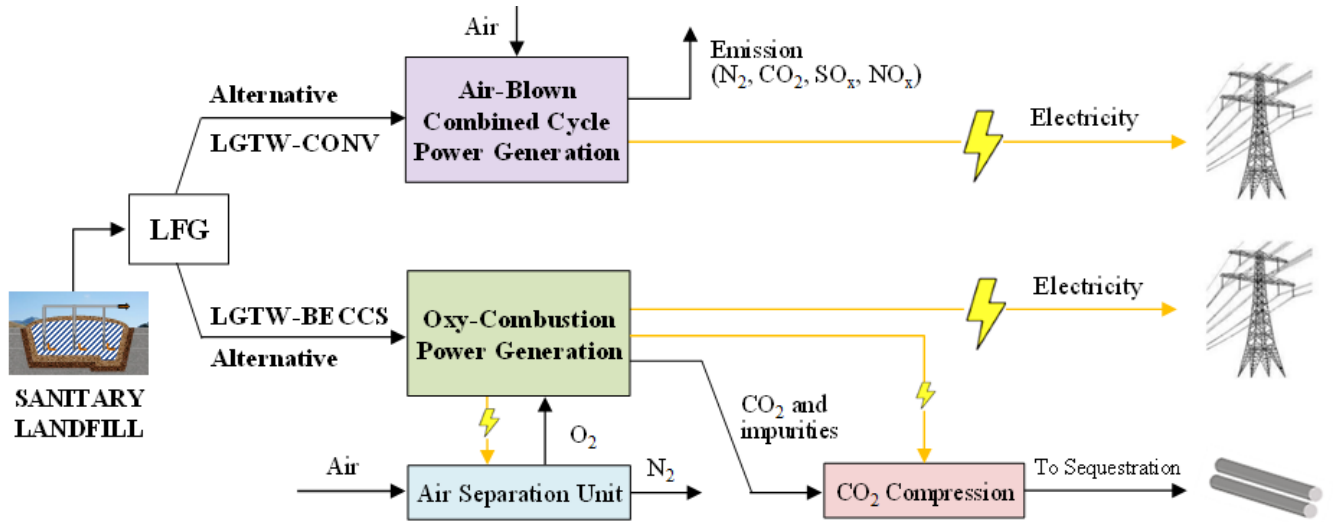
2 Assumptions for simulation, design and economic analysis of landfill-gas power plants are
3 presented. Pre-screening of alternatives determines the base-case process of each concept: the
4 oxyfuel LGTW-BECCS and its CO₂-emitting conventional counterpart LGTW-CONV.

5 **2.1. Simulation of Landfill-Gas-To-Wire Alternatives**

6 Fig. 1 depicts a diagram of LGTW-CONV and LGTW-BECCS wherein operating conditions
7 are investigated. All LGTW solutions were simulated in Aspen-HYSYS v8.8 according to
8 assumptions in Table 1 and have several aspects in common: (i) as landfill-gas may inflame
9 at high temperatures due to O₂ intrusion from air, it is compressed through multistage
10 intercooled compressor train ($T^{Max}=150^{\circ}C$) to feed the GT; (ii) all LGTW's are combined-
11 cycles with HRSG fed with GT exhausts generating the same superheated steam ($T=560^{\circ}C$,
12 $P=70bar$) to the Rankine-Cycle; (iii) single-pressure Rankine-Cycle to reduce investment and
13 because high exhaust temperatures weakens the need for multiple steam pressure levels [39];
14 (iv) ST discharges at 0.10 bar allowing ordinary cooling-water (CW) in the condenser.

15 Moreover, all LGTW's consume $\approx 1.08MMSm^3/d$ of landfill-gas at atmospheric pressure with
16 $\approx 50\%$ mol CH₄ in dry-basis. Landfill-gas composition (Table 1, {A2}) is in agreement with
17 the EPA ranges [13]. H₂S content of ≈ 100 ppm-mol is chosen accordingly to data available
18 elsewhere [46]. Large-scale electricity generation is assumed for cost-effectiveness [16],
19 since large-scales favor high investments like combined-cycles and cryogenic oxygen
20 production. Through 20 years of a large-scale hypothetical landfill, the bulk of the collected
21 landfill-gas is assumed supplied at constant flow rate to the LGTW. The scope for the
22 comparison of different LGTW solutions does not comprehend: (i) the upstream chain
23 aspects (costs and CO₂ emissions included) of landfill-gas supply; (ii) possibilities regarding

- 1 minor parallel landfill-gas utilizations (e.g., heating); and (iii) possibly unnecessary landfill-
- 2 gas pre-purification steps (e.g., siloxanes removal).



3
4 **Figure 1. Landfill-Gas-To-Wire routes (LFG=landfill-gas).**

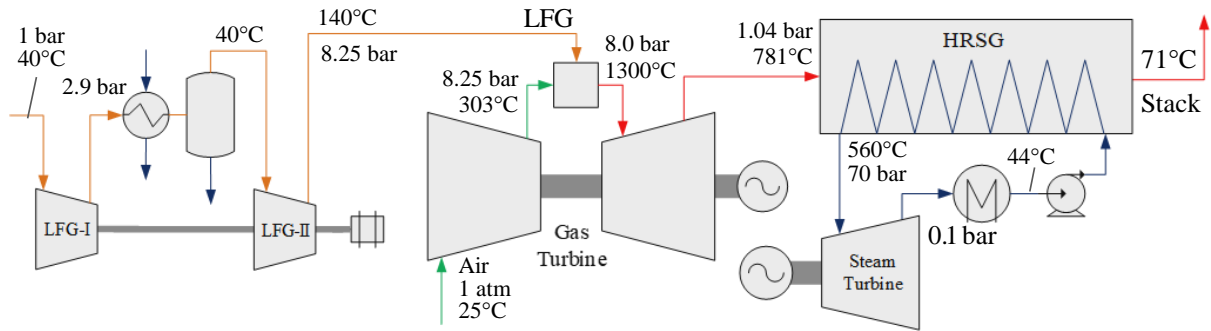
5 **Table 1. Simulation assumptions for LGTW alternatives.**

Item	Assumption
{A1}	Thermodynamic Modeling: Peng-Robinson Equation-of-State, ASME-Table for Steam-Cycle
{A2}	Landfill-Gas (%mol): $F=1869 \text{ kmol/h}$ ($1.08\text{MMSm}^3/\text{d}$), $T=40^\circ\text{C}$, $P=1 \text{ atm}$, $46.37\%\text{CH}_4$, $44.51\%\text{CO}_2$, $1.85\%\text{N}_2$, $0.01\%\text{H}_2\text{S}$, $7.26\%\text{H}_2\text{O}$ (water-saturated)
{A3}	Oxygen (%mol): 1832 kmol/h ($1.41 \cdot 10^6 \text{ kg/d}$), $T=15^\circ\text{C}$, $P=1 \text{ atm}$, $95\%\text{O}_2$, $2.39\%\text{Ar}$, $2.61\%\text{N}_2$
{A4}	ASU Specific Power Consumption: $E_{sep}=139\text{kWh/tO}_2$ [32]
{A5}	Combined-Cycle: 1:1 (Gas-Turbine:Steam-Turbine)
{A6}	Adiabatic Efficiencies: $\eta^{\text{Expander}}=90\%$; $\eta^{\text{Axial-Compressor}}=85\%$; $\eta^{\text{Centrifugal-Compressor}}=80\%$; $\eta^{\text{Pump}}=75\%$.
{A7}	Gas-Turbine (GT) Expander: $T^{\text{Inlet}}=1300^\circ\text{C}$.
{A8}	Single-Pressure Steam-Turbine (ST): $T^{\text{Inlet}}=560^\circ\text{C}$; $P^{\text{Inlet}}=70\text{bar}$.
{A9}	Vacuum-Condenser: $P^{\text{Inlet}}=0.10 \text{ bar}$, $\Delta P^{\text{Head-Loss}}=1 \text{ kPa}$, $T^{\text{Outlet}}=43.8^\circ\text{C}$.
{A10}	HRSG: $\Delta T^{\text{Approach}} \geq 20^\circ\text{C}$, $\Delta P^{\text{GAS}}=3 \text{ kPa}$, $\Delta P^{\text{H}_2\text{O}}=50 \text{ kPa}$.
{A11}	Flue-Gas Direct-Contact Column: Structured-Packing, Theoretical-Stages=3; Recycled-Water: 35°C , $P^{\text{TOP}}=1\text{atm}$, $\Delta P=2\text{kPa}$
{A12}	Cooling-Water (CW): $T^{\text{CW-INLET}}=30^\circ\text{C}$, $T^{\text{CW-OUTLET}}=40^\circ\text{C}$.
{A12}	Intercoolers: $T^{\text{GAS}}=40^\circ\text{C}$, $\Delta P=3\%P^{\text{Inlet}} \leq 50\text{kPa}$.
{A13}	CO_2 : $T^{\text{Liquefaction}}=40^\circ\text{C}$; $P^{\text{Liquefaction}}=150\text{bar}$; $P^{\text{Exportation}}=250\text{bar}$.
{A14}	Landfill-Gas Upstream/Parallel Processes: Not Evaluated

6

1 2.1.1. Landfill-Gas-To-Wire Conventional Process

2 Fig. 2 presents LGTW-CONV flowsheet with economically optimal values. Compressed
3 landfill-gas feeds a conventional air-blown GT whose exhausts feed the HRSG.

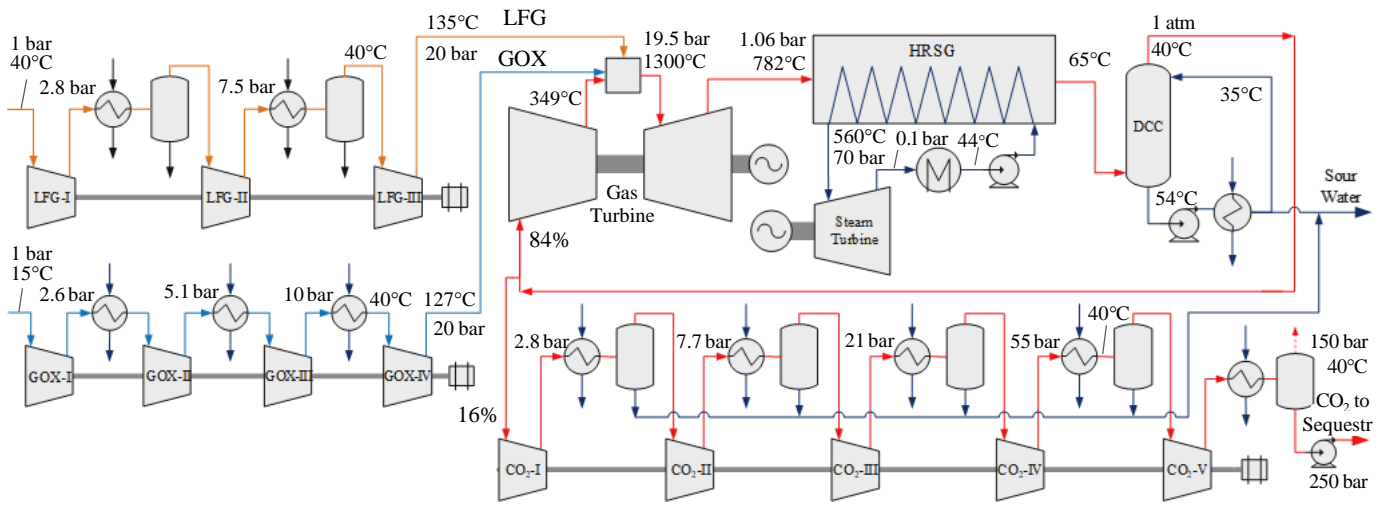


4
5 **Figure 2. LGTW-CONV Base-Case.**

6 2.1.2. Landfill-Gas-To-Wire Oxyfuel Process

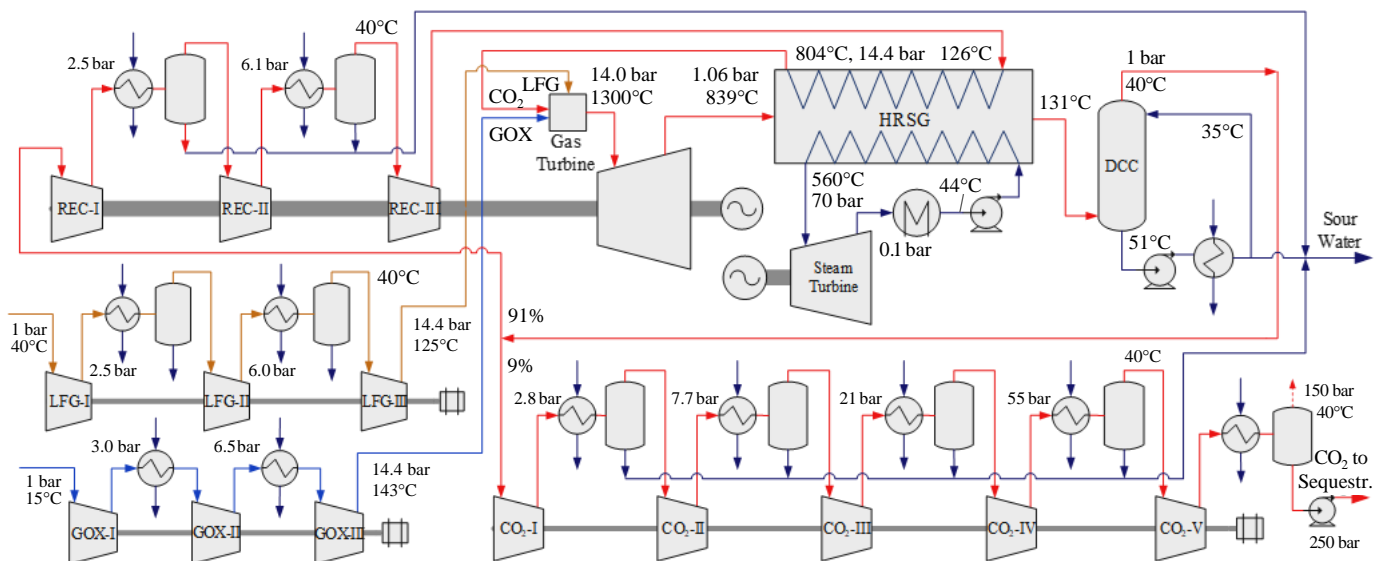
7 Two configurations are considered for oxyfuel LGTW-BECCS: (i) semi-closed oxy-
8 combustion combined-cycle (SCOC-CC) [35], with single-stage adiabatic CO₂-recycle
9 compression (Fig. 3); and (ii) regenerative intercooled oxy-combustion combined-cycle
10 (RIOC-CC), with multistage intercooled CO₂-recycle compression (Fig. 4). RIOCC
11 resembles the E-Matiant cycle without GT reheat [47], incorporating a bottoming Rankine-
12 Cycle. SCOC-CC and RIOCC share several aspects in common: (i) a standalone TVR-
13 2REB ASU [32] supplies atmospheric oxygen at stoichiometric proportion; (ii) oxygen and
14 landfill-gas have multistage intercooled compressor trains ($T^{Max}=150^{\circ}C$) to feed the GT,
15 wherein, multistage intercooled oxygen compression guarantees safety, minimal investment
16 and better process sustainability by avoiding high-pressure hot O₂; (iii) after the HRSG, flue-
17 gas is cooled in a direct-contact column (DCC) with recirculated condensate at the top [39];
18 (iv) part of the DCC top (CO₂-rich) gas is the CO₂ product sent to a multistage intercooled
19 compressor train to be dispatched as 250 bar dense fluid for CCS, the rest recycles to the GT.
20 Fig. 3 presents the SCOC-CC version of LGTW-BECCS with economically optimal values.

- 1 Part of the DCC CO₂-rich gas recycles to GT combustion chamber to abate combustion
- 2 temperature [37] through the single-stage adiabatic axial compressor driven by GT shaft.



3 **Figure 3. SCOC-CC LGTW-BECCS Base-Case (LFG=landfill-gas, GOX=gas-O₂).**

- 4
- 5
- 6 Fig. 4 depicts RIOCC at economically optimal conditions. Regenerative Brayton-Cycle
- 7 and intercooled recycle compression are considered (similarly to E-Matiant cycle) to
- 8 investigate whether lower power demand and reduced size of CO₂-recycle compressors
- 9 improves profitability. The main differences to SCOC-CC comprehend a larger CO₂-recycle
- 10 stream passing through the intercooled centrifugal-compressor train and subsequently
- 11 through the HRSG to be heated to ≈804°C for injection in the GT chamber.



12 **Figure 4. RIOCC LGTW-BECCS Base-Case (LFG=landfill-gas, GOX=gas-O₂).**

13

1 **2.2. Economic Analysis of Landfill-Gas-To-Wire Processes**

2 The economic assessment of LGTWs is performed via Turton et al. [48]. *FCI* is calculated
 3 with equipment sizes via Campbell [49]. GT *FCI* (FCI_{GT}) is estimated adding compressor,
 4 expander and generator *FCI* contributions. As commercial GTs for CO₂-rich fluids are still
 5 inexistent, the same GT *FCI* correlations are used in LGTW-CONV and LGTW-BECCS.
 6 Economic equations are in Supplement S1, Supplementary Materials. Table 2 presents
 7 economic assumptions. Considering that a zero-emission oxyfuel LGTW-BECCS can be
 8 more profitable than LGTW-CONV under CO₂ taxation, it is desired to find the required tax
 9 level to make the net present value (*NPV*) of LGTW-BECCS greater than that of LGTW-
 10 CONV after 20 operation years. Thus, economic performances are evaluated under several
 11 taxation scenarios. Landfill operation related emissions – e.g., fugitive emissions, flare gas,
 12 other landfill-gas uses – were not accounted. Similarly, the analysis does not cover solid-
 13 waste handling, collection, transportation, and processing steps.

14 **Table 2. Economic assumptions.**

Item	Assumption
{E1}	Electricity=0.1087 USD/kWh (USA-Price June/2017).
{E2}	Base-Scenario CO ₂ Taxation: 0 USD/kg
{E3}	Cost of Utilities (<i>CUT</i>): $CW=0.016$ USD/t
{E4}	Equipment <i>FCI</i> : extrapolated with 0.6 exponent if out of correlation ranges [48]
{E5}	ASU <i>FCI</i> : extrapolated with 0.5 exponent from $FCI=141$ MMUSD for 52 kg/s Oxygen [50]
{E6}	<i>FCI</i> Inflation Factor: $CEPCI=603.1$ (2018-average)
{E7}	Construction: three years (20%/30%/50% investment allocations)
{E8}	Operation: 8000 h/y
{E9}	Annual Depreciation: 10% <i>FCI</i>
{E10}	Income Tax Rate: $ITR=34\%$
{E11}	Project Horizon: 20 operation years
{E12}	Annual Interest Rate: $i=10\%$
{E13}	Landfill-Gas Price: zero

15

1 **3. Results and Discussion**

2 Sec. 3.1 addresses techno-economic evaluation of LGTW-CONV variants to seek most
3 profitable operating conditions for the landfill-gas (Table 1, {A2}). Sec. 3.2 compares the
4 performance of the two LGTW-BECCS configurations and similarly evaluates their variants
5 for the best configuration and operating conditions. Sec. 3.3 consolidates the comparison of
6 energy/economic performances of base-cases LGTW-CONV and LGTW-BECCS
7 considering CO₂ taxation.

8 **3.1. Conventional Landfill-Gas-To-Wire Variants**

9 Five LGTW-CONV variants are evaluated; each one featuring a different P_{Comb} . Detailed
10 conditions of LGTW-CONV variants #1 to #5 are found in Supplement S2, Supplementary
11 Materials. Breakdown of machinery contributions to overall power output is presented in
12 Table 3, clearly revealing the expected trade-off between GT and ST powers from different
13 GT outlet temperatures. Raising GT pressure-ratio up to ≈ 20 improves power generation,
14 with smaller gains being attained from 16 to 20 bar due to negative contribution of landfill-
15 gas compressors (Table 3). The corresponding CO₂ emission-factors (Table 3) range from
16 0.711 to 0.753 kg/kWh from different power capacities. Changing Rankine-Cycle conditions
17 to more complex schemes – with multiple pressure levels and reheat – can improve Rankine-
18 Cycle efficiency for reduced GT outlet temperatures (i.e., at higher pressure-ratios), allowing
19 slightly greater net power outputs, which are offset by the respective greater investments
20 from a *NPV* perspective. Therefore, this aspect is ignored for searching the best possible
21 *NPV*.

22 Although conventional natural gas combined-cycles generally operate with $P_{Comb} \approx 20$ bar
23 [39], a GT with reduced pressure-ratio is found to be more suitable for this application, which
24 is unveiled via a long-term *NPV* viewpoint. In this sense, techno-economic comparison of

1 LGTW-CONV variants #1 to #5 is consolidated in Table 4 and Fig. 5. Table 4 presents power
2 output, revenues (*REV*), manufacturing cost (*COM*), *FCI*, and 20 years *NPV* (without CO₂
3 tax) as functions of P_{Comb} , with the corresponding numbers of landfill-gas compression
4 stages. *REV* expresses gains with power output (Table 2, {E1}), while *COM* combines
5 utilities and labor costs in addition to *FCI* derived costs (maintenance, insurance, taxes,
6 overhead and administration). *FCI* and *COM* increase considerably at greater P_{Comb} , while
7 *REV* improvement becomes gradually smaller. Fig. 5 illustrates net efficiency and *NPV*-
8 20years, showing that despite maximum efficiency occurs at $P_{Comb} \approx 20$ bar (variant #5) the
9 highest *NPV*-20years is nearby $P_{Comb} \approx 8$ bar, thus indicating variant #2 as LGTW-CONV
10 Base-Case for further comparisons with LGTW-BECCS. Appendix A (Figs. A.1a to A.1c)
11 presents $NPV \div NPV_{max}$ dependence upon P_{Comb} for LGTW-CONV in scenarios of interest
12 rate, operating hours and electricity price, all showing small influence over optimal P_{Comb} .

13 **Table 3. Power contributions and CO₂ emission-factor of LGTW-CONV variants.**

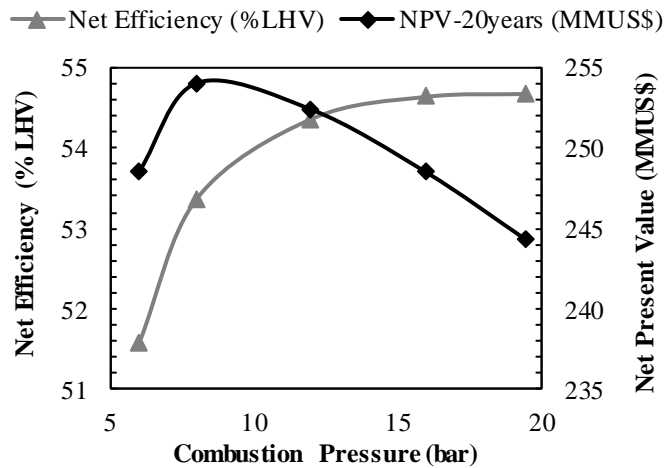
LGTW-CONV Variant	#1	#2*	#3	#3	#4
Combustion Pressure (bar)	6.0	8.0	12.0	16.0	19.5
Landfill-Gas Compressor (MW)	-3.35	-3.93	-4.56	-5.11	-5.49
Gas-Turbine (MW)	55.27	62.11	70.30	75.33	78.47
Steam-Turbine (MW)	48.12	45.30	39.63	35.66	32.94
Rankine-Cycle Pump (MW)	-0.38	-0.35	-0.31	-0.28	-0.26
Overall Output (MW)	99.67	103.13	105.06	105.59	105.66
CO₂ Emission-Factor (kg/kWh)	0.753	0.728	0.715	0.711	0.711

14 * LGTW-CONV Base-Case

15 **Table 4. LGTW-CONV techno-economic sensitivity analysis.**

LGTW-CONV Variant	P_{Comb} (bar)	#Stages landfill-gas	Net Power (MW)	<i>REV</i> (MMUSD/y)	<i>COM</i> (MMUSD/y)	<i>FCI</i> (MMUSD)	<i>NPV</i>-20y (MMUSD)
1	6.0	2	99.67	86.67	16.17	79.1	248.6
2*	8.0	2	103.1	89.68	17.08	84.6	254.0
3	12.0	3	105.1	91.35	18.14	91.4	252.0
4	16.0	3	105.6	91.81	18.86	96.0	248.5
5	19.5	3	105.7	91.87	19.40	99.4	244.3

16 * LGTW-CONV Base-Case



1
2 **Figure 5. Influence of P_{Comb} on LGTW-CONV net efficiency and $NPV-20years$.**

3
4 **3.2. Oxy-Combustion Variants**

5 Ten oxyfuel LGTW-BECCS variants are compared on energy/economic grounds, either
6 adopting SCOC-CC (variants #1 to #5) or RIOCC-CC (variants #6 to #10) configuration at
7 different P_{Comb} . Detailed conditions of LGTW-BECCS variants #1 to #10 are in Supplement
8 S3, Supplementary Materials. LGTW-BECCS variants generate power without CO₂
9 emissions and sequestering 74.7 tCO₂/h. Table 5 presents machinery contributions to overall
10 power output of variants.

11 SCOC-CC has lower GT power than RIOCC-CC – 69.62 against 103.8 MW – mainly
12 accounting for reduced CO₂-recycle flow rate and higher specific CO₂-recycle compression
13 power. In variants #3 and #7 ($P_{Comb} \approx 20$ bar), the specific CO₂-recycle power demands are
14 306 and 264 kJ/kg respectively, but flow rate differences entail total demands of 35.52 and
15 52.32 MW, with respective expander powers of 105.14 and 156.08 MW.

1 **Table 5. Power contribution of process machinery: LGTW-BECCS variants.**

LGTW-BECCS Variant	#1	#2	#3 [†]	#4	#5	#6	#7	#8	#9	#10
Combustion Pressure (bar)	8.0	14.0	19.5	29.5	39.5	14.0	19.5	29.5	39.5	59.5
Configuration	SCOC-CC					RIOCC-CC				
Air Separation Unit (MW)	-7.74	-7.74	-7.74	-7.74	-7.74	-7.74	-7.74	-7.74	-7.74	-7.74
Oxygen Compressor (MW)	-3.84	-4.97	-5.54	-6.37	-6.97	-4.97	-5.54	-6.37	-6.97	-7.66
Fuel-Gas Compressor (MW)	-3.93	-4.86	-5.49	-6.12	-6.64	-4.86	-5.49	-6.11	-6.64	-7.38
Gas-Turbine (MW)	52.39	63.59	69.62	76.46	80.92	102.0	103.8	106.8	108.1	109.5
Steam-turbine (MW)	50.59	46.51	44.28	40.81	37.89	13.43	12.16	10.56	9.41	7.86
CO ₂ Compressor/Pump(MW)	-9.42	-9.42	-9.42	-9.42	-9.42	-9.41	-9.41	-9.41	-9.41	-9.41
Auxiliary Equipment (MW)	-0.43	-0.40	-0.38	-0.36	-0.34	-0.16	-0.15	-0.13	-0.12	-0.11
Overall Output (MW)	77.63	82.71	85.33	87.26	87.71	88.25	87.60	87.57	86.69	85.04
CO₂ Ratio (kg/kWh)	0.962	0.903	0.875	0.856	0.852	0.846	0.853	0.853	0.862	0.878

2 [†] LGTW-BECCS Base-Case.

3 The proportion between GT and ST outputs differs considerably from LGTW-CONV in both
4 LGTW-BECCS configurations (notably RIOCC-CC). In SCOC-CC, due to low isentropic
5 exponent ($\gamma=c_p/c_v$) of CO₂-rich fluid, the GT pressure-ratio needs to be shifted to ≈ 40 to attain
6 usual ST power proportion of 1/3 of total power (Table5). In RIOCC-CC, ST has a minor role –
7 $\approx 1/10$ of GT power – since the exhaust gas mostly heats CO₂-recycle instead of condensate.
8 This concentrates power generation within the regenerative Brayton-Cycle, leaving a small
9 duty to Rankine-Cycle; i.e., superheated steam can optionally be used in one or two ST
10 drivers due to low capacity (e.g., driving landfill-gas and CO₂ compressors, saving $4 \cdot 10^6$
11 USD of *FCI* in variant #6).

12 Table 5 shows that P_{Comb} has opposite effects over SCOC-CC and RIOCC-CC performances
13 within the evaluated ranges. The highest net efficiency was evinced in RIOCC-CC (45.67%
14 LHV), whose output is favored by reduced P_{Comb} of 14 bar (88.25 MW), because more heat is
15 available to increase CO₂-recycle temperature (GT exhaust at $T \approx 839^\circ\text{C}$). Further pressure
16 reduction in RIOCC-CC is limited by reduced GT power and increased losses in heat
17 exchange. SCOC-CC, on the other hand, produces more power with P_{Comb} increase up to ≈ 40
18 bar (87.71 MW), exhibiting minor gains from 30 to 40 bar. Further P_{Comb} increase in SCOC-

1 CC is limited by increased power consumption of CO₂-recycle adiabatic compression, as CO₂
2 becomes too hot for compression.

3 **3.2.1. Comparison with Literature**

4 It is worthwhile to indicate similarities of SCOC-CC simulation results with literature works,
5 even if slightly different design conditions apply (notably T_{Comb}). Dahlquist et al. [39] results
6 showed similar SCOC-CC net efficiency dependence with GT pressure-ratio and presented a
7 curve that coincidentally flattens out at $P_{Comb} \approx 40$ bar for $T_{Comb} = 1340^\circ\text{C}$. Assuming higher
8 T_{Comb} , Yang et al. [43] investigated the effect of higher pressure-ratios beyond the point of
9 efficiency reduction, finding the maximum output in the middle of flat efficiency curves at
10 ≈ 60 for $T_{Comb} = 1418^\circ\text{C}$, and at ≈ 90 for $T_{Comb} = 1600^\circ\text{C}$. As small power gains occurred for
11 P_{Comb} from 40 to 60 bar at $T_{Comb} = 1418^\circ\text{C}$, operation with $P_{Comb} \approx 40$ bar should be more
12 suitable in view of other *FCI* aspects. Table 5 indicates that, from power generation
13 perspective, the appropriate P_{Comb} for application in SCOC-CC LGTW-BECCS should be
14 between 30-40 bar. Indeed, for T_{Comb} within 1300-1400°C, the SCOC-CC GT pressure-ratio
15 ranges typically from 30 to 40; slightly below the maximum output condition via simulation
16 [34]. The development of high pressure-ratio machines would hardly be motivated by small
17 efficiency gains, as discussed by Dahlquist et al. [39] regarding defying GT designs for
18 pressure-ratios above 40 even for conventional GTs, and becoming even more challenging
19 for CO₂-rich fluid (e.g., low sound speed). Thus, for applications with $T_{Comb} = 1340^\circ\text{C}$ and GT
20 outlet temperature of 620°C , $P_{Comb} \approx 32$ bar was selected in [39].

21 **3.2.2. Oxyfuel Optimal Combustion Pressure**

22 Contrasting with abovementioned references, determination of the best P_{Comb} is here
23 approached for maximal *NPV*. Although high GT pressure-ratios are generally conceived for
24 SCOC-CC aiming at efficiency optimization, the viewpoint of long-term *NPV* reveals that

1 usual $P_{Comb} \approx 20$ bar is suitable for LGTW-BECCS application, mainly due to a major role of
 2 FCI in NPV analysis. Table 6 consolidates techno-economic comparison of LGTW-BECCS
 3 variants #1 to #10 and presents power output, REV , COM , FCI , $NPV-20years$ and numbers of
 4 compressor stages (for CO_2 -recycle, oxygen and landfill-gas) as functions of P_{Comb} . As in
 5 LGTW-CONV variants, depending on whether SCOC-CC or RIOCC-CC is adopted, greater
 6 P_{Comb} entails considerable increase of FCI and COM , while REV behaves proportionally to
 7 power output.

8 **Table 6. LGTW-BECCS techno-economic sensitivity analysis.**

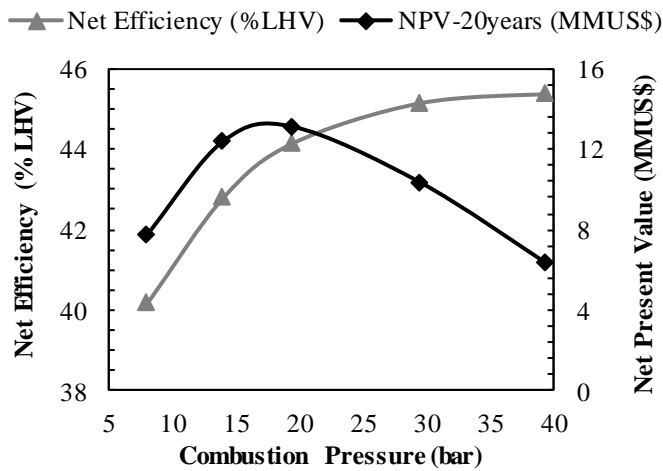
Process Variant	CC	P^{Comb} (bar)	#Stages Recycle	#Stages landfill-gas	#Stages O_2	Net Power (MW)	REV (MMUSD/y)	COM (MMUSD/y)	FCI (MMUSD)	$NPV-20y$ (MMUSD)
1	SCOC	8.0	1	2	3	77.63	67.52	37.31	193.1	7.81
2		14.0	1	3	3	82.71	71.94	39.11	203.6	12.39
3 [†]		19.5	1	3	4	85.33	74.21	40.25	210.2	13.08
4		29.5	1	4	4	87.26	75.88	41.50	217.4	10.35
5		39.5	1	4	4	87.71	76.27	42.22	221.6	6.36
6	RIOC	14.0	3	3	3	88.25	76.75	44.90	237.3	-12.68
7		19.5	3	3	4	87.60	76.17	45.12	237.7	-16.24
8		29.5	4	4	4	87.57	76.15	46.81	247.1	-29.40
9		39.5	4	4	4	86.69	75.38	46.79	247.0	-32.37
10		59.5	4	4	4	85.04	73.95	47.08	248.5	-40.60

9 [†] LGTW-BECCS Base-Case.

10
 11 Albeit presenting slightly lower net efficiency, SCOC-CC is much more profitable than
 12 RIOCC-CC, which, in turn, was proved economically unfeasible at the end of the horizon due
 13 to significantly higher FCI . Therefore, as RIOCC-CC performs economically worse,
 14 inefficiently exploiting landfill-gas potential, from this point on only SCOC-CC is considered
 15 for LGTW-BECCS.

16 Fig. 6 depicts sensitivity analysis for SCOC-CC net efficiency and $NPV-20years$ dependence
 17 upon P_{Comb} . The net efficiency curve flattens out around the maximum as similarly showed
 18 elsewhere [39]. Despite the maximum efficiency at $P_{Comb} \approx 40$ bar (variant #5), NPV shows
 19 that the highest profitability is nearby $P_{Comb} \approx 20$ bar, thus indicating variant #3 as LGTW-

1 BECCS Base-Case for comparison with LGTW-CONV Base-Case. Appendix A (Figs. A.1d
 2 to A.1f) presents $NPV \div NPV_{max}$ dependence upon P_{Comb} for SCOC-CC LGTW-BECCS in
 3 scenarios of interest rate, operating hours and electricity price, all showing small influence
 4 over optimal P_{Comb} , and confirming $P_{Comb} \approx 20$ bar for best long-term profitability.



5
 6 **Figure 6. Influence of P_{Comb} on LGTW-BECCS net efficiency and $NPV-20years$.**

7 Considering other oxyfuel LGTW-BECCS candidates, few other cycles appear to be suitable,
 8 making SCOC-CC a competitive LGTW-BECCS configuration. Water-injection oxyfuel
 9 processes worth investigation in future work because, despite of their lower efficiency [37],
 10 expensive CO_2 -recycles are dismissed entailing lower FCI and cost-effectiveness. In contrast,
 11 competitiveness of efficient Graz-cycles is hampered due to process complexity. The Allam-
 12 cycle has efficiency and a simpler configuration – resembling E-Matiant flowsheet without
 13 bottoming-cycle – but is based on high-pressure supercritical CO_2 cycle ($P_{Comb}=300$ bar)
 14 operating at high-pressure GT discharge (≈ 30 bar, $775^\circ C$) negatively impacting FCI [51].

15 **3.3. Performance Comparison of Landfill-Gas-To-Wire Concepts**

16 Performances of LGTW-CONV using conventional combined-cycle (variant #2) and LGTW-
 17 BECCS using SCOC-CC configuration (variant #3) are technically/economically compared
 18 at their respective optimal P_{Comb} for maximum $NPV-20years$. Considering a conceivable

1 BECCS version of LGTW-CONV coupled to post-combustion capture by aqueous-
2 monoethanolamine absorption (PCC-MEA), it is techno-economically demonstrated in
3 Appendix B the conditional superiority of oxyfuel LGTW-BECCS over LGTW-
4 CONV+PCC-MEA.

5 **3.3.1. Performances of Processes**

6 Table 7 presents main streams of LGTW-CONV and LGTW-BECCS regarding Figs. 2 and 3.
7 LGTW-CONV emissions reach 75.05 tCO₂/h accompanied by 0.0111 tSO₂/h, while zero-
8 emission LGTW-BECCS produces 1837 kmol/h of high-pressure CO₂-rich dense fluid
9 (92.45% molCO₂, 4.48% molN₂, 2.38% molAr, 0.36% molO₂, 0.33% molH₂O). Some N₂ and Ar
10 are carried to the CO₂ fluid thanks to their presence in landfill-gas and in the (95% mol) O₂
11 stream (Table 1, {A2,A3}). Higher %CO₂, reducing CO₂-rich fluid compression costs, can be
12 attained via higher purity O₂ supply – up to 98% mol for reasonable overall efficiency – at the
13 expense of slightly higher E_{sep} [32]. Nevertheless, for strict CO₂ purity, further purification of
14 CO₂-rich fluid is necessary.

15 Fig. 3 reveals lower CO₂-recycle to GT of 84% vis-à-vis typical ≈90% [34]. This has to do
16 with the high %CO₂ of landfill-gas (Table 1, {A2}), as the landfill-gas CO₂ enters the
17 combustor at much lower temperature than CO₂-recycle lowering the required CO₂-recycle
18 flow rate. LGTW-BECCS with $P_{Comb} \approx 40$ bar, the optimal power output case (variant#5,
19 Table 6), has slightly higher CO₂-recycle (≈85%) due to its higher CO₂-recycle temperature
20 (442°C) comparatively to 349°C in LGTW-BECCS Base-Case (variant#3, Table 6).

21 Regarding GT exhaust temperature, some studies indicate that it is close to the exhaust
22 temperature of air-blown combined-cycle at optimal pressure-ratio [34]. This is confirmed in
23 Table 7, which presents similar GT outlet temperature for LGTW-CONV and LGTW-
24 BECCS. The achieved high exhaust temperature (≈782°C), caused by low P_{Comb} , is also an

1 outcome of steam-cycle assumptions (Table 1), as advanced Rankine-Cycles favor lower GT
2 temperatures due to more efficient heat exchange, entailing higher optimum P_{Comb} . As
3 discussed in [39], the higher the GT outlet temperature, the lower the need for multiple
4 pressure levels in Rankine-Cycle. More heat above the boiler pinch-point produces more
5 steam; and, as a result of increased water flows, more heat is extracted below the pinch-point.
6 Thus, depending on steam-cycle configuration, the HRSG flue-gas temperature typically
7 varies from 65°C to 130°C [39]. This explains the low temperature of HRSG flue-gas in
8 LGTW-CONV and LGTW-BECCS (Figs. 2 and 3). Lower flue-gas temperature is attained in
9 LGTW-BECCS (65°C), indicating better heat recovery. This is caused by higher fall of fluid
10 specific heat capacity (c_p) through HRSG – 1.32→0.98 kJ/kg.K in LGTW-BECCS against
11 1.25→1.07 kJ/kg.K in LGTW-CONV – implying lower heat recovery in the economizer, thus
12 lower detachment of temperature profiles along HRSG end section.

13 **Table 7. Conditions and molar composition of main streams.**

<i>Stream</i>	LGTW-CONV				LGTW-BECCS							
	<i>Air Inlet</i>	<i>Comb. Outlet</i>	<i>GT Outlet</i>	<i>Stack Gas</i>	<i>ST Inlet</i>	<i>O₂ Feed</i>	<i>Comb. Outlet</i>	<i>GT Outlet</i>	<i>DCC Top</i>	<i>ST Inlet</i>	<i>Sour Water</i>	<i>CO₂</i>
<i>T</i> (°C)	25.0	1300	781	70.9	560	15.0	1300	782	40.0	560	35.4	61.2
<i>P</i> (bar)	1.013	8.00	1.043	1.013	70.0	1.013	19.5	1.060	1.013	70.0	1.50	250
<i>F</i> (kmol/h)	17254	19034	19034	19034	7482	1832	13699	13699	12089	7312	1747	1836
<i>Y_k</i> (molfrac.)												
CO ₂ :	0.0004	0.0896	0.0896	0.0896	-	-	0.7589	0.7589	0.8599	-	0.0004	0.9245
H ₂ O :	0.0189	0.1106	0.1106	0.1106	1.000	-	0.1817	0.1817	0.0729	1.000	0.9996	0.0033
O ₂ :	0.2055	0.0953	0.0953	0.0953	-	0.9500	0.0029	0.0029	0.0033	-	0.0000	0.0036
Ar :	0.0091	0.0083	0.0083	0.0083	-	0.0239	0.0196	0.0196	0.0222	-	0.0000	0.0238
N ₂ :	0.7660	0.6982	0.6982	0.6982	-	0.0261	0.0368	0.0368	0.0417	-	0.0000	0.0448

14
15 From power results in Tables 4 and 6, the gross combined-cycle power output is ≈3.4%LHV
16 (6.49 MW) higher in LGTW-BECCS, but oxygen production and compression consume
17 4.00%LHV and 2.87%LHV, respectively; while CO₂ compression demand further
18 4.87%LHV (Tables 5 and 7). LGTW-BECCS net efficiency (including ASU consumption) is

1 44.16%LHV, hence electricity generation is 9.20%LHV (17.78 MW) lower than LGTW-
2 CONV Base-Case counterpart. This efficiency penalty assumes an efficient TVR-2REB ASU
3 consuming $E_{sep}=139$ kWh/tO₂ [32] and is consistent with the expected penalty of 8-11%LHV
4 from oxyfuel GT works [34]. Since conventional double-column ASUs typically require
5 $E_{sep}=200$ kWh/tO₂ [52], if such ASUs replace TVR-2REB in LGTW-BECCS, ≈ 3.7 MW less
6 power would be produced, further reducing overall efficiency by 1.9%LHV.

7 **3.3.2. Environmental Performances**

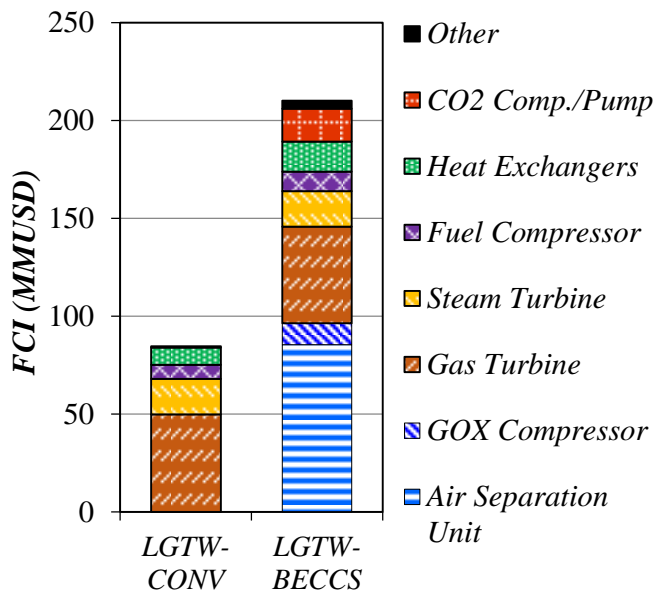
8 Assuming that biomass is the carbon source of 50% of landfill-gas CO₂ [5], LGTW-BECCS
9 drains up to 37.36 tCO₂/h from atmosphere through the carbon cycle. In contrast, LGTW-
10 CONV has limited mitigation potential, besides emitting SO₂, as greenhouse-gas emissions
11 are reduced from 425.90 t/h of CO₂-equivalent in collected landfill-gas (GWP_{100} basis) to
12 75.05 tCO₂/h (stack). Since H₂S contents as high as 3000 ppm-mol can be found in landfill-
13 gas, SO₂ generation may be 30 times higher, intensifying environmental impacts of LGTW-
14 CONV (e.g., acid rain). On the other hand, sour-water production is an issue of LGTW-
15 BECCS, as SO₂ is almost totally dissolved in water effluents from DCC and CO₂
16 compressors knockout-vessels (≈ 94 g/L solubility at 25°C, 1 atm). Such sour-waters can be
17 treated by adding Ca(OH)₂ suspensions [53] precipitating insoluble CaSO₃ for landfilling or
18 for sale.

19 **3.3.3. Economic Performances**

20 Fig. 7 presents *FCI* comparison of LGTW-CONV and Oxyfuel LGTW-BECCS including
21 TVR-2REB ASU. Fig. 7 clearly shows that a major shortcoming of LGTW-BECCS is the
22 high *FCI* of cryogenic ASU – $1.41 \cdot 10^3$ tO₂/d capacity – together with the O₂ compressor
23 train counterpart. LGTW-BECCS also involves landfill-gas compression to the oxyfuel GT at
24 high pressure-ratio, besides CO₂-rich fluid compression to dispatch and a greater number of

1 intercooler heat exchangers. Hence, not surprisingly, the *FCI* of LGTW-BECCS (≈ 210.2
 2 MMUSD) is $\approx 150\%$ higher than LGTW-CONV *FCI* (≈ 84.6 MMUSD).

3 Fig. 7 also unveils similar ≈ 65 MW GT *FCI* (≈ 50 MMUSD) and ≈ 45 MW ST *FCI* (≈ 18
 4 MMUSD). These are reasonable GT/ST values according to Jaramillo and Matthews [20]
 5 data for 1-40 MW GT and 0.5-15 MW ST. Using log-extrapolation and *CEPCI* correction,
 6 *FCI* for 65 MW GT and 45 MW ST become 869 USD/kW (≈ 56 MMUSD) and 288 USD/kW
 7 (≈ 19 MMUSD), respectively. This confirms adequacy and $\pm 20\%$ accuracy of Turton et al.
 8 [48] methods used here.



9
 10 **Figure 7. *FCI* of LGTW-CONV and LGTW-BECCS Base-Cases.**

11 Table 8 shows economic indicators – gross annual profit (*GAP*), annual profit (*AP*), *COM*
 12 and *NPV-20years* – of LGTW-CONV and LGTW-BECCS with TVR-2REB ASU for several
 13 carbon taxations, indicating that both concepts have positive *NPV-20years*. Without CO₂ tax,
 14 LGTW-CONV has evidently greater *NPV-20years* (254 vs 13 MMUSD) due to lower *FCI*
 15 and greater *AP* (51 vs 30 MMUSD/y) which results from higher *REV* and lower *COM*. The
 16 *REV* of LGTW-BECCS (74 MMUSD/y) is lower in the same proportion of power output (-

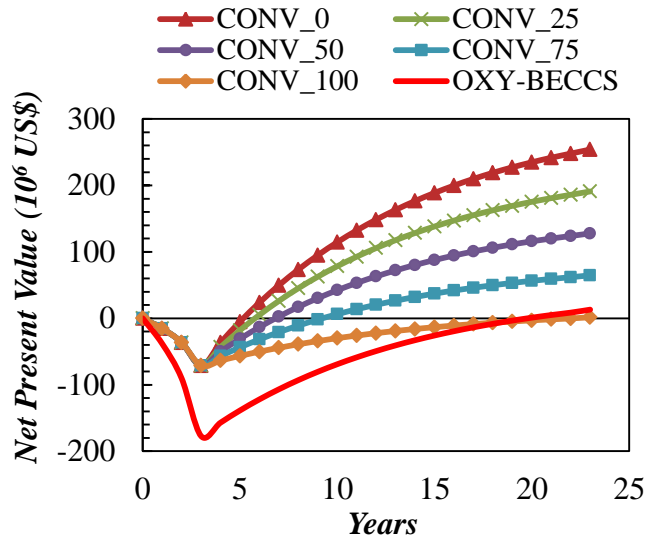
1 17%) and its *COM* is higher due to its 150% higher *FCI*. LGTW-BECCS also has higher cost
 2 of utilities (*CUT*) from higher CW utilization, though with small influence over *COM*.

3 **Table 8. Economic performance: LGTW alternatives under CO₂ taxes.**

Power Plant CO ₂ Tax (USD/t)	LGTW-CONV					LGTW-BECCS
	0	25	50	75	100	(any)
<i>FCI</i> (MMUSD)	84.62	84.62	84.62	84.62	84.62	210.21
<i>REV</i> (MMUSD/y)	89.68	89.68	89.68	89.68	89.68	74.21
<i>CRM</i> (MMUSD/y)	0.00	0.00	0.00	0.00	0.00	0.00
<i>CUT</i> (MMUSD/y)	1.15	1.15	1.15	1.15	1.15	1.60
<i>COM</i> (MMUSD/y)	17.08	32.03	46.98	61.93	76.88	40.25
<i>GAP</i> (MMUSD/y)	72.60	57.65	42.70	27.75	12.80	33.96
<i>AP</i> (MMUSD/y)	50.79	40.92	31.06	21.19	11.32	29.56
<i>NPV-20years</i> (MMUSD)	254.0	190.9	127.8	64.70	1.58	13.08

4

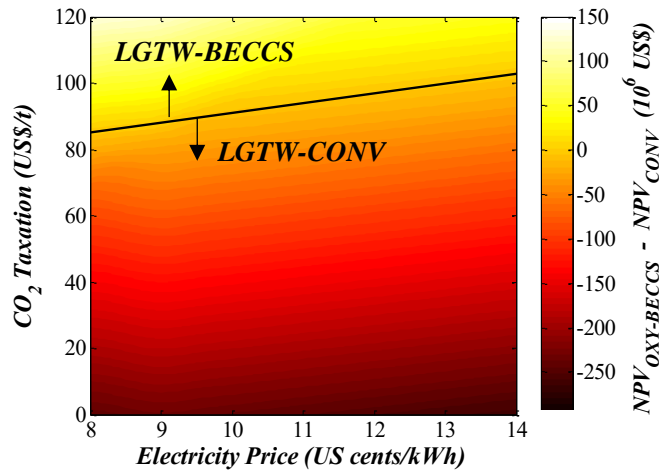
5 These results are reasonable since higher costs and investments are inherent to CCS solutions.
 6 Such economic LGTW-BECCS disadvantage against conventional CO₂-emitting LGTW-
 7 CONV should be offset by carbon reduction policies (e.g., emission taxation) or CO₂
 8 monetization whenever possible (e.g., EOR). Therefore, despite of CCS costs, LGTW-
 9 BECCS can overcome the CO₂-emitting LGTW-CONV in terms of profitability as shown in
 10 Fig. 8, which compares *NPV* versus years under five CO₂ taxation scenarios – 0, 25, 50, 75,
 11 100 USD/t – where the first three years account for plant construction with 20/30/50% capital
 12 outlay. In Fig. 8 five curves belong to LGTW-CONV and a sole curve corresponds to
 13 LGTW-BECCS as it performs zero-emission power generation. Bending of LGTW-CONV
 14 *NPV* profiles, caused by increasing CO₂ taxes, allows the zero-emission LGTW-BECCS to
 15 progressively approximate the LGTW-CONV economic performance through the years. By
 16 the 20th year from construction, LGTW-BECCS overcomes LGTW-CONV for taxes above
 17 95.45 USD/t CO₂.



1
2 **Figure 8. NPV profiles of LGTW-BECCS with TVR-2REB ASU and LGTW-CONV**
3 **under carbon taxation scenarios (USD/tCO₂).**
4

5 Without CO₂ taxes, Fig. 8 shows that 3 years of operation would be sufficient for the payback
6 of LGTW-CONV. As gas-fired combined-cycles entails higher power outputs than simple
7 gas-fired Rankine-Cycles with steam-turbines, one can find a faster payback than the 5 years
8 of Puente Hills project [22]. However, incidence of CO₂ taxation naturally delays LGTW-
9 CONV payback, with medium-term (5-10 years) to long-term (10+ years) campaigns being
10 required above 75 USD/tCO₂ (Fig. 8) due to reduced *AP* (Table 8). At 75 USD/tCO₂ LGTW-
11 BECCS outperforms LGTW-CONV in terms of *AP*, but its huge *FCI* (Fig. 7) hampers a
12 superior *NPV*.

13 Fig. 9 depicts the sensitivity analysis of LGTW-BECCS *NPV* excess to LGTW-CONV as a
14 function of electricity price and CO₂ tax. Fig. 9 indicates the most profitable solution for each
15 region, unveiling superior performance of LGTW-BECCS within ≈85-102 USD/tCO₂ tax
16 range for variations of ≈25% around the assumed base-price (0.1087 USD/kWh). Albeit high
17 for current standards, similar taxations are already seen in European countries, and it is
18 plausible that such taxation levels become extensively employed in incoming decades.



1
2 **Figure 9. Difference $NPV_{OXY-BECCS} - NPV_{CONV}$ versus CO₂ tax and electricity price.**

3
4 **4. Conclusions**

5 Competitiveness of zero-emission LGTW-BECCS concept based on oxyfuel landfill-gas-
6 fired combined-cycle is demonstrated against CO₂-emitting LGTW-CONV charged by CO₂
7 taxation. A scenario of large-scale landfill-gas stable supply is considered. Two different
8 LGTW-BECCS oxyfuel configurations using CO₂-recycle are evaluated and economically
9 compared: SCOC-CC and RIOCC-CC, the former with single-stage adiabatic compression of
10 CO₂-recycle, and the latter with intercooled compression and preheating of CO₂-recycle.
11 RIOCC-CC allows slightly greater net efficiency comparatively to SCOC-CC but at the
12 expenses of inferior NPV , so SCOC-CC configuration is indicated to LGTW-BECCS.

13 Sensitivity analysis on P_{Comb} showed that the highest NPV in LGTW-CONV and LGTW-
14 BECCS via SCOC-CC is attained at ≈ 8.0 bar and ≈ 20 bar, respectively. These relatively low
15 GT pressure-ratios derive from investment and compression requirements for landfill-gas
16 processing. At such conditions, the oxyfuel efficiency penalty is 9.2%LHV with CO₂ capture
17 ratio of 0.875 kg/kWh (Table 5). Comparison of NPV performances along project years under
18 different CO₂ taxation scenarios is provided. Superior profitability of proposed zero-emission
19 LGTW-BECCS over LGTW-CONV exists for ≈ 95 USD/t CO₂ tax.

1 This work contributes to the literature at evaluating the zero-emission LGTW-BECCS
2 concept as sustainable power generation allowing net removal of CO₂ from atmosphere – as
3 urban wastes have reasonable biomass content – making LGTW-BECCS an reliable tool for
4 climate-change mitigation. The proposed concept is demonstrated to be economically feasible
5 without government subsidies and electricity overpricing, attaining break-even price of 0.104
6 USD/kWh. LGTW-BECCS fed by 1.08 MMSm³/d landfill-gas, generates 85.33 MW, the
7 equivalent demand of ≈70,000 US average homes. Therefore, besides presenting remarkable
8 environment performance – particularly against greenhouse-gas emissions – LGTW-BECCS
9 is evinced as a viable and sustainable waste monetization solution that entails economic
10 growth and health-social benefits.

11 **Supplementary Materials**

12 Supplements S1, S2, S3 and S4 are found in the Supplementary Materials available online.

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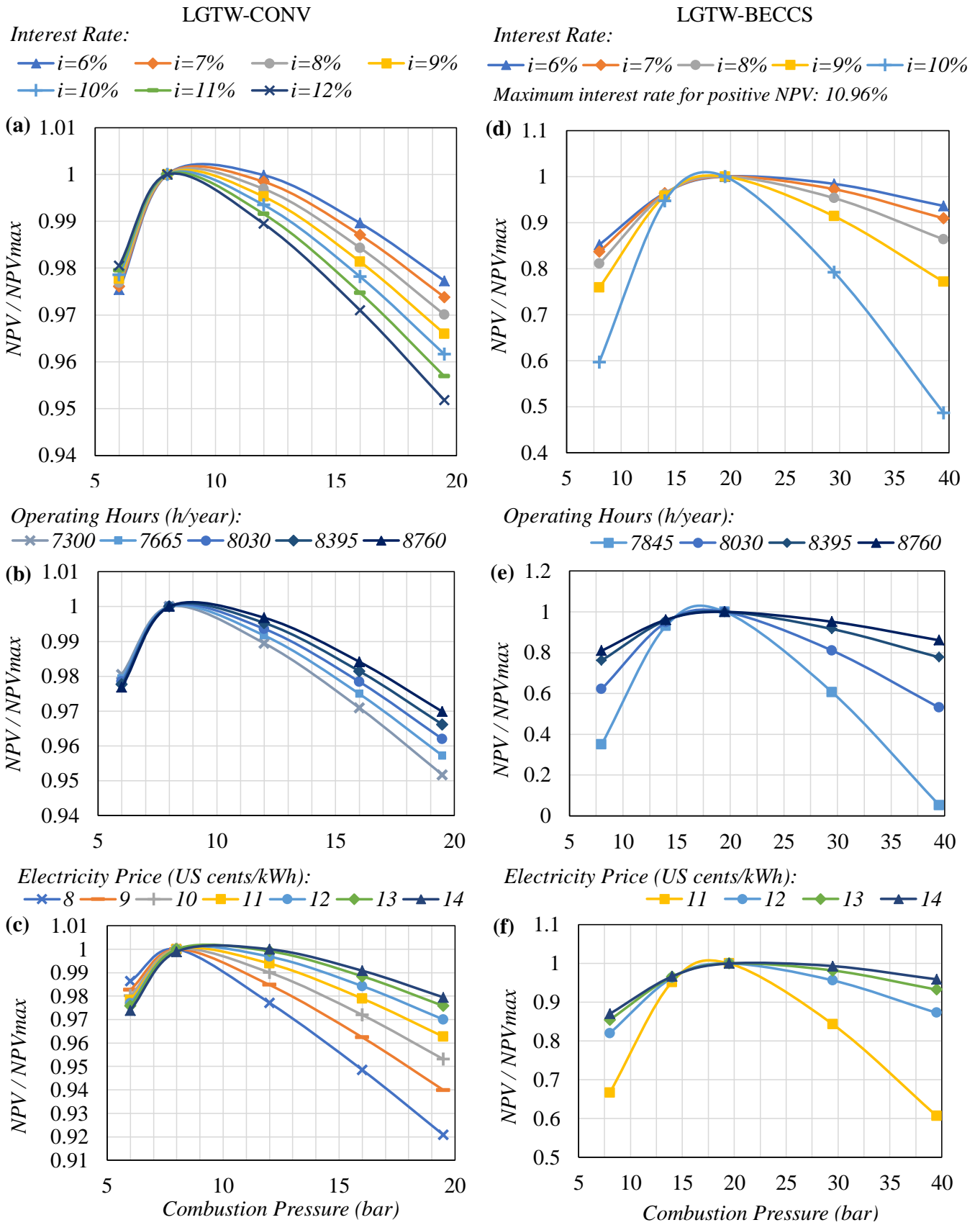
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29

1 **Appendix A: Effect of Economic Parameters on Optimal Combustion Pressure**



2 **Figure A.1. Influence of Interest Rate, Operating Hours and Electricity Price over the**
 3 **$NPV \div NPV_{max}$ dependence upon P_{Comb} for LGTW-CONV (a-c) and LGTW-BECCS (d-f).**

4

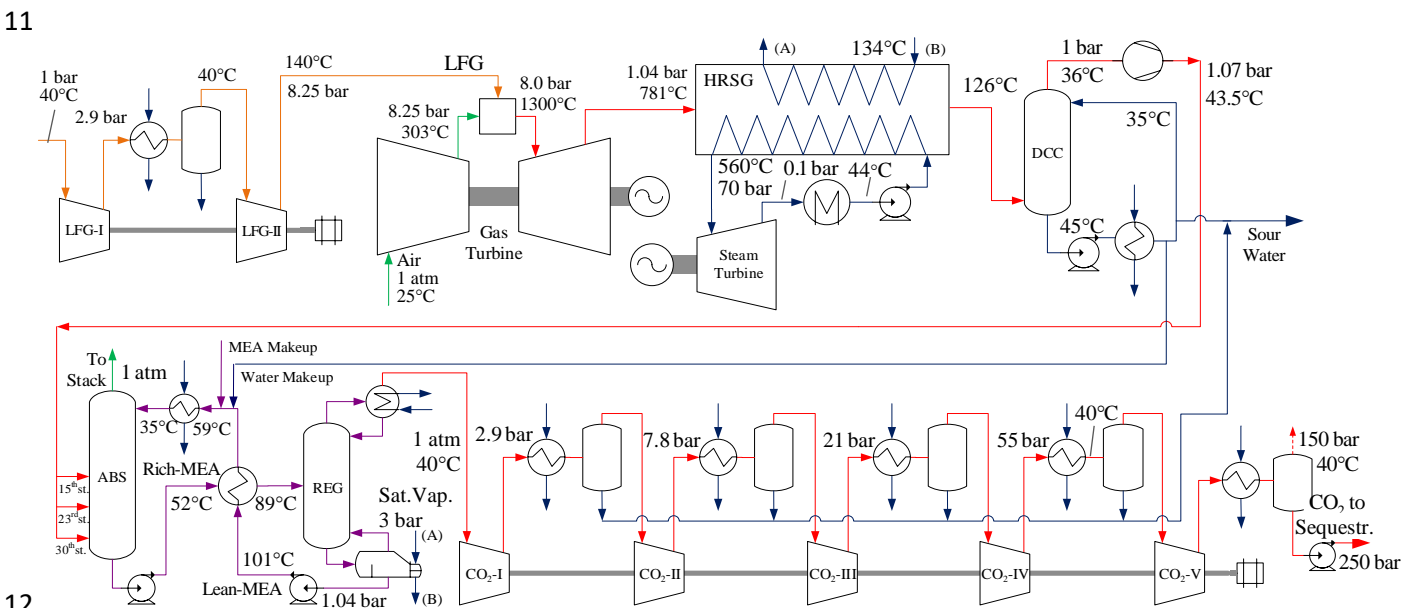
5

1 **Appendix B: Conventional Landfill-Gas-to-Wire with Post-Combustion CO₂ Capture**

2 Techno-economic evaluation of a BECCS version of LGTW-CONV coupled to post-
 3 combustion CO₂ capture via aqueous-monoethanolamine absorption (PCC-MEA) is
 4 performed aiming to unveil oxyfuel LGTW-BECCS superiority over LGTW-CONV+PCC-
 5 MEA as BECCS solutions. Table B.1 lists LGTW-CONV+PCC-MEA technical assumptions.
 6 Fig. B.1 presents LGTW-CONV+PCC-MEA with operating conditions. Detailed conditions
 7 of LGTW-CONV+PCC-MEA are found in Supplement S4, Supplementary Materials.
 8 Extraction of low-pressure steam from HRSG for PCC-MEA solvent regeneration is the only
 9 change required in the LGTW-CONV.

10 **Table B.1. Technical assumptions: LGTW-CONV+PCC-MEA.**

Item	Assumption
{B1}	PCC-MEA Thermodynamic Modeling: Aspen-HYSYS Acid-Gas Package
{B2}	Solvent: Aqueous-Monoethanolamine 30%w/w
{B3}	Capture-efficiency=90%
{B4}	Absorber: Capture-Ratio=20.1 kg ^{Solvent} /kg ^{CO₂inlet} , Structured-Packing (Mellapak-250X), Theoretical-Stages=30, Gas-Feed-Stages={15, 23, 30}, D=8m; P ^{TOP} =1atm, ΔP=6kPa
{B5}	Regenerator: Heat-Ratio=4.03 MJ/kg ^{CO₂captured} , Structured-Packing (Mellapak-250X), Theoretical-Stages=15, Feed-Stage=7, D=4.5m; P ^{TOP} =1atm, ΔP=3kPa
{B6}	Reboiler Utility: Low-Pressure Saturated-Steam from HRSG, T=134°C



12 **Figure B.1. LGTW-CONV+PCC-MEA (ABS=Absorber, REG=Regenerator).**

1 Flue-gas from HRSG is cooled down to 36°C in DCC and blown from 1.00 to 1.07 bar to
2 feed PCC-MEA at three different inlets to improve capture [44]. The CO₂-rich solvent is pre-
3 heated before feeding the regeneration column. Hot lean solvent leaves the regenerator as
4 bottoms and water-saturated CO₂ (1 atm, 40°C) is the top-condenser gas product. The CO₂
5 loadings of Lean-MEA and Rich-MEA are 0.300 and 0.504 mol^{CO₂}/mol^{MEA}, respectively.
6 Captured CO₂ is compressed to 250 bar for dense fluid dispatch as in the oxyfuel LGTW-
7 BECCS. The CO₂ flow rate is 67.61 t/h (99.5% molCO₂). Table B.2 presents machinery
8 contributions to overall power output and CO₂ emission-factor of LGTW-CONV+PCC-
9 MEA. Results are compared to LGTW-CONV and LGTW-BECCS Base-Cases (Tables 3 and
10 5). The available heat to Rankine-Cycle is reduced thanks to HRSG supply of low-pressure
11 steam for PCC-MEA solvent regeneration (4.03 MJ/kg^{CO₂captured}), entailing a drastic fall in ST
12 output from 45.30 to 14.91 MW. Such high CCS penalty is an outcome of high CO₂ content
13 in the landfill-gas. Compression of CO₂ further rises the CCS penalty (7.74 MW), though less
14 impacting than oxyfuel LGTW-BECCS (9.42 MW, Table 5) due to higher CO₂ purity and
15 inferior capture-efficiency. Hence, LGTW-CONV+PCC-MEA loses power output from
16 103.13 to 63.93 MW (Tables 3 and B.2), and is outperformed by oxyfuel LGTW-BECCS
17 output (85.33 MW, Table 5). PCC-MEA reduces the CO₂ emission-factor of LGTW-CONV
18 from 0.728 to 0.120 kg/kWh (Tables 3 and B.2), being still susceptible to carbon charges.

19 **Table B.2. Power contributions and emission-factor: LGTW-CONV+PCC-MEA.**

Item	LGTW + PCC-MEA
Landfill-Gas Compressor (MW)	-3.93
Gas-Turbine (MW)	62.11
Steam-Turbine (MW)	14.91
Rankine-Cycle Pump (MW)	-0.12
Flue-Gas Blower (MW)	-1.14
CO ₂ Compressor/Pump(MW)	-7.74
Auxiliary Equipment (MW)	-0.17
Overall Output (MW)	63.93
CO₂ Emission-Factor (kg/kWh)	0.120

20

1 Table B.3 presents economic performance of LGTW-CONV+PCC-MEA for several carbon
2 taxes, attaining positive *NPV-20years* for taxes below 100 USD/tCO₂. In terms of *FCI*, the
3 low-emission LGTW-CONV+PCC-MEA overcomes oxyfuel LGTW-BECCS (141.87 vs
4 210.21 MMUSD). Consequently, without CO₂ tax, LGTW-CONV+PCC-MEA has also
5 greater *NPV-20years* (26 vs 13 MMUSD) despite of its lower *AP* (22.6 vs 29.6 MMUSD/y).
6 The corresponding break-even electricity price for positive *NPV-20years* is 0.097 USD/kWh.
7 Lower *REV* results, following inferior power output (-25%). The *COM* is lower as a result of
8 inferior *FCI*, despite of monoethanolamine raw materials cost for makeup (0.59 MMUSD/y).

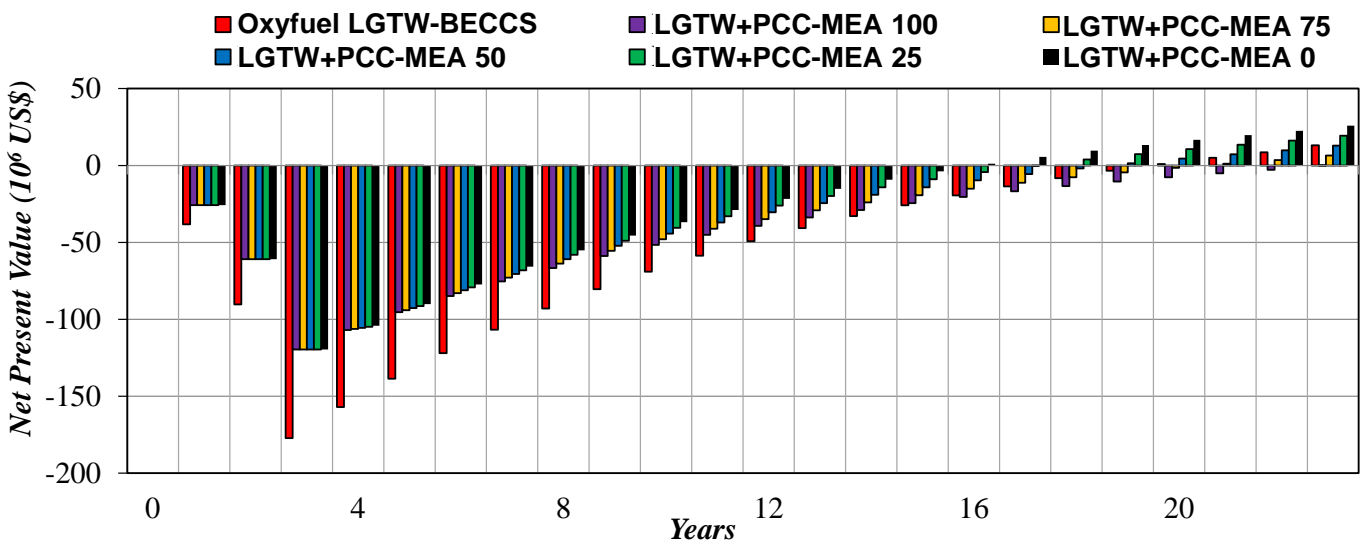
9 **Table B.3. Economic performance under CO₂ taxes: LGTW-CONV+PCC-MEA .**

CO ₂ Tax (USD/tCO ₂)	LGTW-CONV+PCC-MEA				
	0	25	50	75	100
<i>FCI</i> (MMUSD)	141.87	141.87	141.87	141.87	141.87
<i>REV</i> (MMUSD/y)	55.60	55.60	55.60	55.60	55.60
<i>CRM</i> (MMUSD/y)	0.00	0.00	0.00	0.00	0.00
<i>CUT</i> (MMUSD/y)	2.17	2.17	2.17	2.17	2.17
<i>COM</i> (MMUSD/y)	28.64	30.17	31.70	33.23	34.76
<i>GAP</i> (MMUSD/y)	26.96	25.43	23.90	22.37	20.84
<i>AP</i> (MMUSD/y)	22.62	21.61	20.60	19.59	18.58
<i>NPV-20years</i> (MMUSD)	25.85	19.40	12.94	6.48	0.02

10

11 Tables B.3 and 8 demonstrate that the proposed zero-emission LGTW-BECCS (*NPV-20years*
12 of 13.08 MMUSD) overcomes LGTW-CONV+PCC-MEA in terms of profitability for CO₂
13 taxes above ≈50 USD/tCO₂. This is also evinced in Fig. B.2 which depicts *NPV* profiles
14 under CO₂ taxation scenarios. Reduced annual profit (*AP*) of LGTW-CONV+PCC-MEA
15 caused by CO₂ taxes allows the zero-emission LGTW-BECCS to progressively surpass its
16 *NPV* through the years. Similarly to LGTW-BECCS, LGTW-CONV+PCC-MEA is also
17 superior to emitting LGTW-CONV (Fig. 8) in long-term *NPV* for taxes above 100.7
18 USD/tCO₂, though with slightly negative *NPV-20years*.

1 Adoption of PCC-MEA implies some sustainability concerns absent in oxyfuel LGTW-
 2 BECCS, such as, some generation of solvent-wastes, occupational issues regarding solvent
 3 proximity, and monoethanolamine steady supply for solvent makeup entailing
 4 costs/storage/logistic issues. Therefore, viewed as two BECCS solutions, oxyfuel LGTW-
 5 BECCS is slight inferior to post-combustion LGTW-CONV+PCC-MEA in terms of *NPV*-
 6 *20years*, but can outperform LGTW-CONV+PCC-MEA economically for taxes above 50
 7 USD/tCO₂ with superior sustainability.



8
 9 **Figure B.2. NPV profiles under carbon taxation (values in USD/tCO₂): zero-emission**
 10 **LGTW-BECCS with TVR-2REB ASU versus low-emission LGTW-CONV+PCC-MEA.**
 11